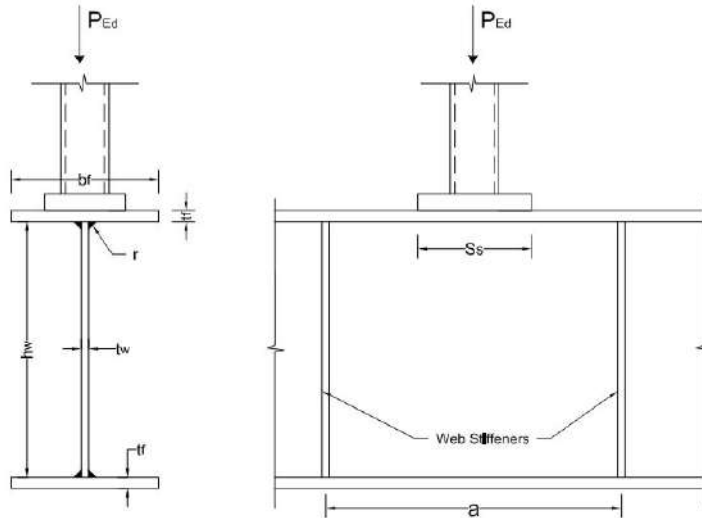


## Beam web under concentrated load according to Eurocode 3/1.5

### INPUT DATA

### COMPUTED OUTPUT

### DATA TO BE CHECKED



Characteristic Yield strength of structural steel of the beam:

$$f_{yk} := 275 \text{ MPa}$$

Characteristic Yield strength of structural steel for the plate:

$$f_{yf} := f_{yk} = 275 \text{ MPa}$$

Characteristic Yield strength of structural steel of the web:

$$f_{yw} := f_{yk} = 275 \text{ MPa}$$

Safety factors

$$\gamma_{M1} := 1$$

$$\gamma_{M0} := 1$$

Design axial compression force of the beam at the examined location

$$N_{Ed} := 15 \text{ kN}$$

Design Bending Moment of the beam at the examined location

$$M_{Ed} := 232 \text{ kN} \cdot \text{m}$$

Transverse force on the beam (downwards)

$$P_{Ed} := 500 \text{ kN}$$

Main beam flange width

$$b_{flange} := 280 \text{ mm}$$

Flange thickness

$$t_f := 13 \text{ mm}$$

Main beam web thickness

$$t_w := 8 \text{ mm}$$

Radius of fillet between flange and web

$$r := 24 \text{ mm}$$

Beam web height

$$h_w := 400 \text{ mm}$$

Length of stiff bearing

$$S_s := 400 \text{ mm}$$

Modulus of Elasticity of steel

$$E_s := 210000 \text{ MPa}$$

Figure.(6.2)

$$S_s := \min(S_s, h_w) = 400 \text{ mm}$$

The length of stiff bearing  $S_s$  on the flange should be taken as the distance over which the applied load is effectively distributed at a slope of 1:1, see Figure 6.2. However,  $S_s$  should not be taken as larger than  $h_w$ .

Distance between adjacent transverse stiffeners

$$a := 500 \text{ mm}$$

$$\varepsilon := \sqrt{\frac{235 \cdot \text{MPa}}{f_{yf}}} = 0.924$$

$$b_f := \min(b_{flange}, 2 \cdot 15 \cdot \varepsilon \cdot t_f) = 280 \text{ mm}$$

## 1. Resistance against concentrated vertical load

### 1.1 Effective loaded length

$$m_1 := \frac{f_{yf} \cdot b_f}{f_{yw} \cdot t_w} = 35 \quad \text{Eq.(6.8)}$$

Assume that  $\lambda_F > 0.5$

$$m_2 := 0.02 \cdot \left(\frac{h_w}{t_f}\right)^2 = 18.935 \quad \text{Eq.(6.9)}$$

Therefore

$$l_y := \min\left((S_s + 2 \cdot t_f \cdot (1 + \sqrt{m_1 + m_2})), a\right) = 50 \text{ cm} \quad \text{effective loaded length} \quad \text{Eq.(6.10)}$$

### 1.2 $k_F$ coefficient

$$k_F := 6 + 2 \cdot \left(\frac{h_w}{a}\right)^2 = 7.28 \quad \text{Figure.(6.1)}$$

### Critical Buckling Load

$$F_{cr} := 0.90 \cdot k_F \cdot E_s \cdot \frac{t_w^3}{h_w} = 1761 \text{ kN} \quad \text{Eq.(6.5)}$$

$$\lambda_F := \sqrt{\frac{l_y \cdot t_w \cdot f_{yw}}{F_{cr}}} = 0.79$$

$$\text{result} := \begin{cases} \text{if } (\lambda_F > 0.5) \\ \quad \left\| \begin{array}{l} m_2 \leftarrow 0.02 \cdot \left(\frac{h_w}{t_f}\right)^2 \\ \text{else} \\ \quad \left\| m_2 \leftarrow 0 \end{array} \right. \\ \text{return } m_2 \end{cases}$$

$$m_2 := \text{result} = 18.935$$

$$l_y := \min\left(\left(S_s + 2 \cdot t_f \cdot \left(1 + \sqrt{m_1 + m_2}\right)\right), a\right) = 50 \text{ cm}$$

$$k_F := 6 + 2 \cdot \left(\frac{h_w}{a}\right)^2 = 7.28$$

$$F_{cr} := 0.90 \cdot k_F \cdot E_s \cdot \frac{t_w^3}{h_w} = 1761 \text{ kN}$$

$$\lambda_F := \sqrt{\frac{l_y \cdot t_w \cdot f_{yw}}{F_{cr}}} = 0.79$$

$$x_F := \min\left(\frac{0.5}{\lambda_F}, 1\right) = 0.633$$

Reduction function

Eq.(6.3)

$$L_{eff} := x_F \cdot l_y = 31.633 \text{ cm}$$

effective length for resistance  
to transverse forces

Eq.(6.2)

### 1.3 Design Resistance to local buckling under transverse forces

$$F_{Rd} := \frac{f_{yw} \cdot L_{eff} \cdot t_w}{\gamma_{M1}} = 695.9 \text{ kN}$$

Eq.(6.1)

if ( $F_{Rd} > P_{Ed}$ , "SAFE", "NOT SAFE") = "SAFE"

$$n_2 := \frac{P_{Ed}}{F_{Rd}} = 0.718$$

Eq.(6.14)

## 2. Check against normal stresses due to Bending and Axial Force acting on the beam

Flange Classification

$$classification\_flange := c\_over\_t \leftarrow \frac{\left(\frac{b_{flange} - t_w}{2} - r \cdot \sqrt{2}\right)}{t_f}$$

EC3/1.1

if ( $c\_over\_t \leq 9 \cdot \epsilon$ )

|| "Category 1"

else if ( $c\_over\_t > 9 \cdot \epsilon \wedge c\_over\_t \leq 10 \cdot \epsilon$ )

|| "Category 2"

else if ( $c\_over\_t > 10 \cdot \epsilon \wedge c\_over\_t \leq 14 \cdot \epsilon$ )

|| "Category 3"

else

|| "Category 4"

Table 5.2 (sheet 2 of 3)

classification\_flange = "Category 1"

If classification of flange is "4", it is advisable to change the design section

#### Web Classification for bending

$$\text{classification\_web\_bending} := \left\| \begin{array}{l} c\_over\_t \leftarrow \frac{(h_w - 2 \cdot r \cdot \sqrt{2})}{t_w} \\ \text{if } (c\_over\_t \leq 72 \cdot \epsilon) \\ \quad \| \text{"Category 1"} \\ \text{else if } (c\_over\_t > 72 \cdot \epsilon) \wedge (c\_over\_t \leq 83 \cdot \epsilon) \\ \quad \| \text{"Category 2"} \\ \text{else if } (c\_over\_t > 83 \cdot \epsilon) \wedge (c\_over\_t \leq 124 \cdot \epsilon) \\ \quad \| \text{"Category 3"} \\ \text{else} \\ \quad \| \text{"Category 4"} \end{array} \right\|$$

Table 5.2 (sheet 1 of 3)

**classification\_web\_bending = "Category 1"**

If classification of web bending is "4", it is advisable to change the design section

#### Web Classification for compression

$$\text{classification\_web\_compression} := \left\| \begin{array}{l} c\_over\_t \leftarrow \frac{(h_w - 2 \cdot r \cdot \sqrt{2})}{t_w} \\ \text{if } (c\_over\_t \leq 33 \cdot \epsilon) \\ \quad \| \text{"Category 1"} \\ \text{else if } (c\_over\_t > 33 \cdot \epsilon) \wedge (c\_over\_t \leq 38 \cdot \epsilon) \\ \quad \| \text{"Category 2"} \\ \text{else if } (c\_over\_t > 38 \cdot \epsilon) \wedge (c\_over\_t \leq 42 \cdot \epsilon) \\ \quad \| \text{"Category 3"} \\ \text{else} \\ \quad \| \text{"Category 4"} \end{array} \right\|$$

Table 5.2  
(sheet 1 of 3)

**classification\_web\_compression = "Category 4"**

$k_{\sigma} := 4$

buckling factor corresponding to the stress ratio  $\psi$  and boundary conditions. For long plates  $k_{\sigma}$  is given in Table 4.1 or Table 4.2

$\psi := 1$

stress ratio determined in accordance with 4.4(3) and 4.4(4)

Table 4.1: Internal compression elements

Stress distribution (compression positive)	Effective <sup>d</sup> width $b_{eff}$
	$\psi = 1$ $b_{eff} = \rho \cdot b$ $b_{e1} = 0,5 \cdot b_{eff}$ $b_{e2} = 0,5 \cdot b_{eff}$
	$1 > \psi \geq 0$ $b_{eff} = \rho \cdot b$ $b_{e1} = \frac{2}{5 - \psi} \cdot b_{eff}$ $b_{e2} = b_{eff} - b_{e1}$
	$\psi < 0$ $b_{eff} = \rho \cdot b_c = \rho \cdot b / (1 - \psi)$ $b_{e1} = 0,4 \cdot b_{eff}$ $b_{e2} = 0,6 \cdot b_{eff}$
$\psi = \sigma_2 / \sigma_1$	1 $1 > \psi > 0$ 0
Buckling factor $k_{\sigma}$	4,0 $8,2 / (1,05 + \psi)$ 7,81
	$7,81 - 6,29\psi + 9,78\psi^2$ -1 $-1 > \psi > -3$
	23,9 $5,98 \cdot (1 - \psi)^2$

Table 4.2: Outstand compression elements

Stress distribution (compression positive)	Effective <sup>d</sup> width $b_{eff}$
	$1 > \psi \geq 0$ $b_{eff} = \rho \cdot c$
	$\psi < 0$ $b_{eff} = \rho \cdot b_c = \rho \cdot c / (1 - \psi)$
$\psi = \sigma_2 / \sigma_1$	1   0   -1 $1 > \psi \geq -3$
Buckling factor $k_{\sigma}$	0,43   0,57   0,85 $0,57 - 0,21\psi + 0,07\psi^2$
	$1 > \psi \geq 0$ $b_{eff} = \rho \cdot c$
	$\psi < 0$ $b_{eff} = \rho \cdot b_c = \rho \cdot c / (1 - \psi)$
$\psi = \sigma_2 / \sigma_1$	1 $1 > \psi > 0$ 0 $0 > \psi > -1$ -1
Buckling factor $k_{\sigma}$	0,43 $0,578 / (\psi + 0,34)$ 1,70 $1,7 - 5\psi + 17,1\psi^2$ 23,8

$$\lambda_p := \frac{(h_w - 2 \cdot r \cdot \sqrt{2})}{t_w} = 0.791$$

$$\rho := \begin{cases} \text{if } (\lambda_p > 0.673) \\ \left| \left| \rho \leftarrow \frac{\lambda_p - 0.055 \cdot (3 + \psi)}{\lambda_p^2} \right. \right| \\ \text{else} \\ \left| \left| \rho \leftarrow 1 \right. \right| \end{cases}$$

Reduction factor  $\rho$  for internal compression elements Eq.(4.2)

$\rho = 0.913$

Effective cross section

$A_{eff} := 2 \cdot b_f \cdot t_f + \rho \cdot h_w \cdot t_w = 102.011 \text{ cm}^2$

Moment of inertia of section

$I_y := \frac{1}{12} \cdot t_w \cdot h_w^3 + 2 \cdot b_f \cdot t_f \cdot \left( \frac{(h_w + t_f)}{2} \right)^2 = 35310 \text{ cm}^4$

## Elastic Section Modulus

$$y_{max} := \left( \frac{h_w + t_f}{2} + \frac{t_f}{2} \right) = 21.3 \text{ cm}$$

$$W_{el.y} := \frac{I_y}{y_{max}} = 1658 \text{ cm}^3$$

$$W_{eff} := W_{el.y} = 1658 \text{ cm}^3$$

## Verification

$$n_1 := \frac{N_{Ed}}{f_{yk} \cdot A_{eff}} + \frac{M_{Ed}}{f_{yk} \cdot W_{eff}} = 0.514 \quad \text{Eq.(4.15)}$$

$\gamma_{M0}$                        $\gamma_{M0}$

if ( $n_1 < 1$ , "SAFE", "NOT SAFE") = "SAFE"

## 3. Interaction between transverse force, bending moment and axial force

if ( $n_2 + 0.8 \cdot n_1 \leq 1.4$ , "SAFE", "NOT SAFE") = "SAFE"                      Eq.(7.2)

## 4. Flange induced buckling

$$A_W := h_w \cdot t_w = 32 \text{ cm}^2$$

is the cross section area of the web

$$A_{fc} := b_f \cdot t_f = 36.4 \text{ cm}^2$$

fc is the effective cross section area of the compression flange

$$k := 0.4$$

plastic moment resistance utilized for the beam resistance

To prevent the compression flange buckling in the plane of the web, the following criterion should be met:

$$\text{if} \left( \frac{h_w}{t_w} \leq k \cdot \frac{E_s}{f_{yf}} \cdot \sqrt{\frac{A_W}{A_{fc}}}, \text{"SAFE"}, \text{"NOT SAFE"} \right) = \text{"SAFE"} \quad \text{Eq.(8.1)}$$

$$\text{ratio} := \frac{\frac{h_w}{t_w}}{k \cdot \frac{E_s}{f_{yf}} \cdot \sqrt{\frac{A_W}{A_{fc}}}} = 0.175$$